

Balancing the Segway

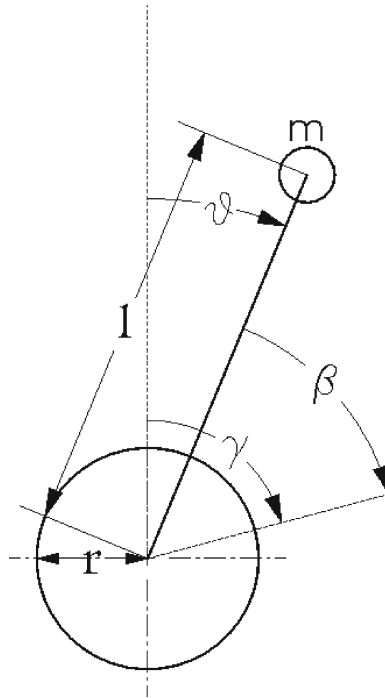
Charles Masenas
cmasenas@alum.rpi.edu
<http://nxtotherway.webs.com>



Objective

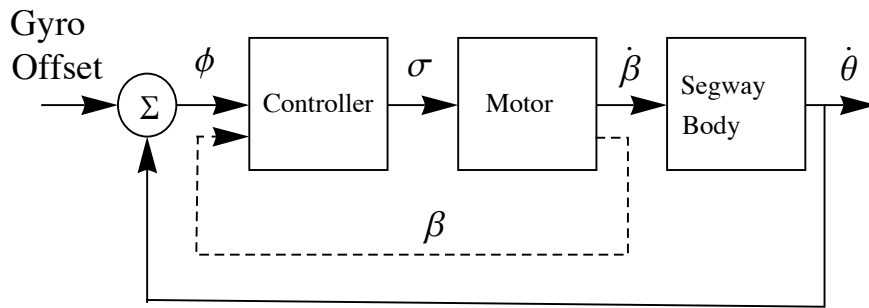
To demonstrate that using available software to solve systems of differential equations, it is not necessary to know elaborate control theory to solve the two wheel balancing problem.

Segway Body



- Point mass, m
- Tilt angle θ from vertical reference
- Wheel angle β from arm reference

Loop Model



■ Strategy for Solution

- Model the Motor and the Segway Body
- Design the Controller

Software Implementation of the Controller

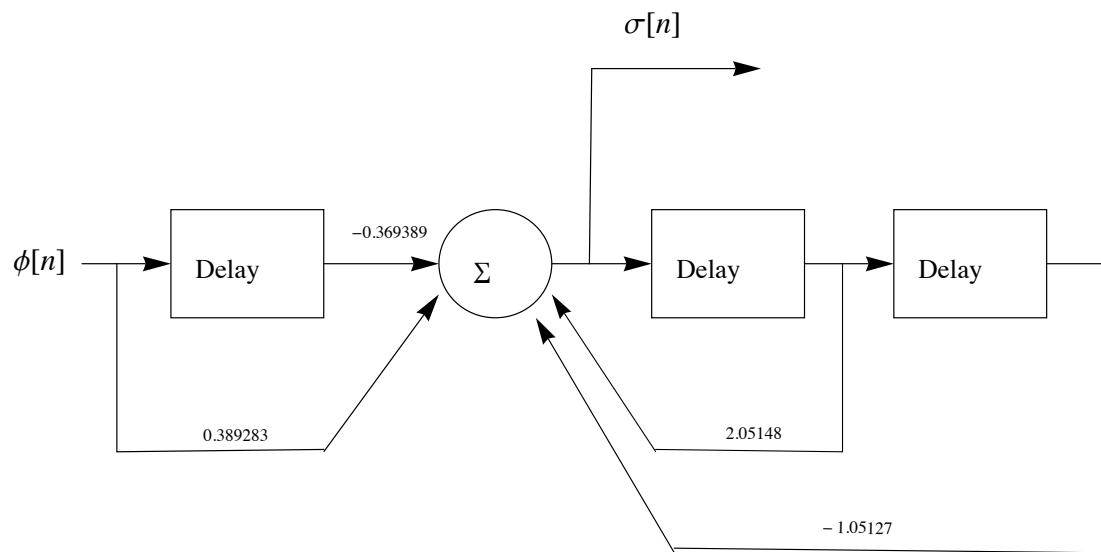
```
while ( balancing )
{
    PHI[2] = ReadGyro( ) ; // read the gyro, tilt angle change per time, deg/sec

    // This line implements the controller as a digital filter
    SIGMA[2] = -0.369389 PHI[0] + 0.389283 PHI[1] - 1.05127 SIGMA[0] + 2.05148 SIGMA[1] ;

    // shift the registers in preparation for the next cycle
    SIGMA[0] = SIGMA[1] ;
    SIGMA[1] = SIGMA[2] ;
    PHI[0] = PHI[1] ;

    SetMotorSpeed( SIGMA[2] );

    Wait( 12 milliseconds);
}
```



5 Minute Frequency Domain Speed Date

Time Domain

Newton's Second Law

$$f(t) = m D^2 x(t)$$

$$\delta(t) \uparrow$$

$$u(t) \text{ (step function)}$$

Frequency Domain

$$F(s) = m s^2 X(s)$$

$$F(s) \rightarrow \boxed{\frac{1}{m s^2}} \rightarrow X(s)$$

$$\rightarrow \boxed{1} \rightarrow \Delta(s)$$

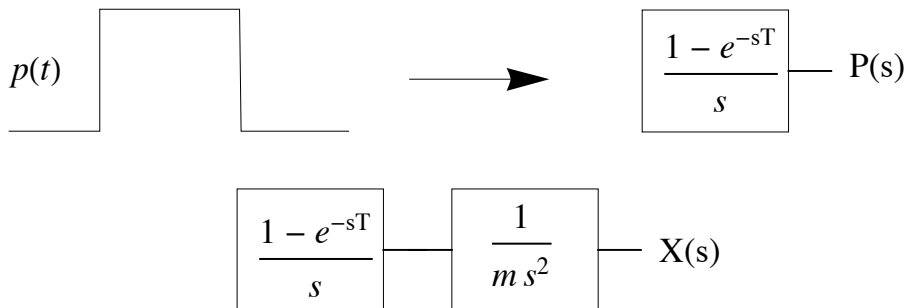
$$\rightarrow \boxed{\frac{1}{s}} \rightarrow U(s)$$

5 Minute Frequency Domain Speed Date (Continued)

$$g(t - T) \longrightarrow G(s) e^{-sT}$$

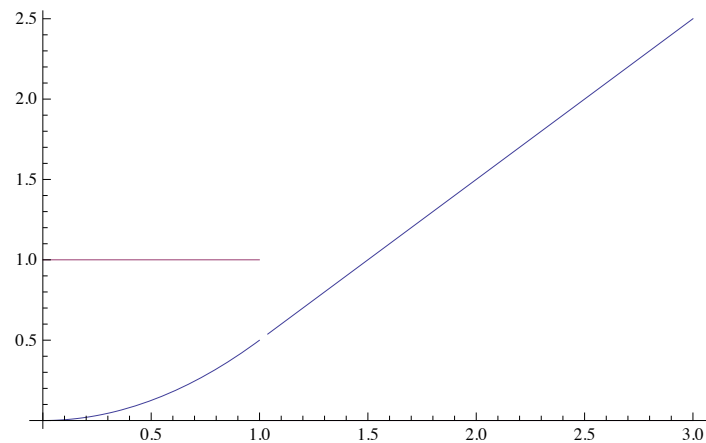
Note: In the discrete-time frequency domain,
 $z \equiv e^{sT}$ is the independent variable.

$$g(nT) \longrightarrow g_n z^{-n}$$

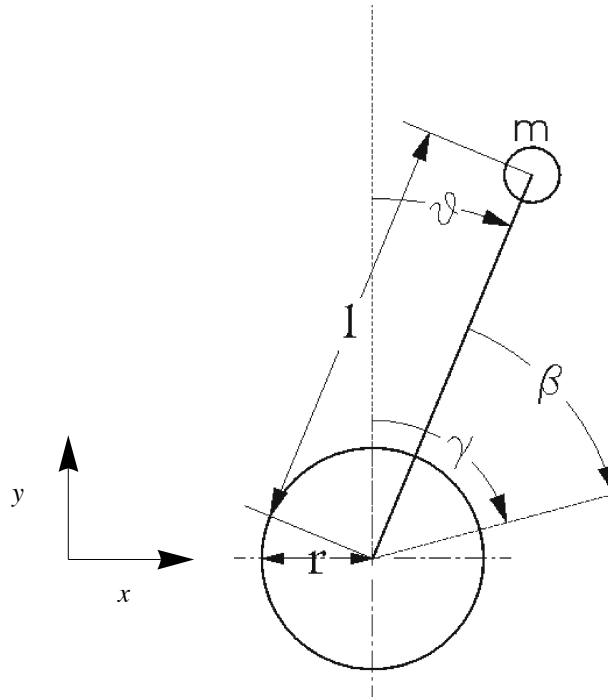


```
In[26]:= Plot[ { InverseLaplaceTransform[  $\frac{1 - E^{-sT}}{s} \frac{1}{s^2}$ , s, t],
  InverseLaplaceTransform[  $\frac{1 - E^{-sT}}{s}$ , s, t] }, {t, 0, 3} ]
```

Out[26]=



Modeling of Segway Body (Lagrangian Mechanics)



Coordinates of point mass “m”...

$$x(t) = r [\theta(t) + \beta(t)] + l \sin(\theta(t))$$

$$y(t) = l \cos(\theta(t))$$

Kinetic energy, T...

$$T = \frac{1}{2} m \left(\left[\frac{d}{dt} x(t) \right]^2 + \left[\frac{d}{dt} y(t) \right]^2 \right)$$

Potential energy, V...

$$V = m g y(t)$$

Modeling of Segway Body (Lagrangian Mechanics)

■ Lagrangian, L

- $L = T - V$

- $$L = \frac{1}{2} m \left([l \dot{\theta}(t) \sin(\theta(t))]^2 + [l \dot{\theta}(t) \cos(\theta(t)) + r (\dot{\beta}(t) + \dot{\theta}(t))]^2 \right) - m g l \cos(\theta(t))$$

■ Lagrangian equation for each coordinate (θ , β).

- $\frac{d}{dt} \frac{\partial L}{\partial \dot{\theta}} - \frac{\partial L}{\partial \theta} = \tau_{\theta}$

- $\frac{d}{dt} \frac{\partial L}{\partial \dot{\beta}} - \frac{\partial L}{\partial \beta} = \tau_{\beta}$

- Torque on arm = - Torque on wheel

- $\frac{d}{dt} \frac{\partial L}{\partial \dot{\theta}} - \frac{\partial L}{\partial \theta} = - \left(\frac{d}{dt} \frac{\partial L}{\partial \dot{\beta}} - \frac{\partial L}{\partial \beta} \right)$

■ Result

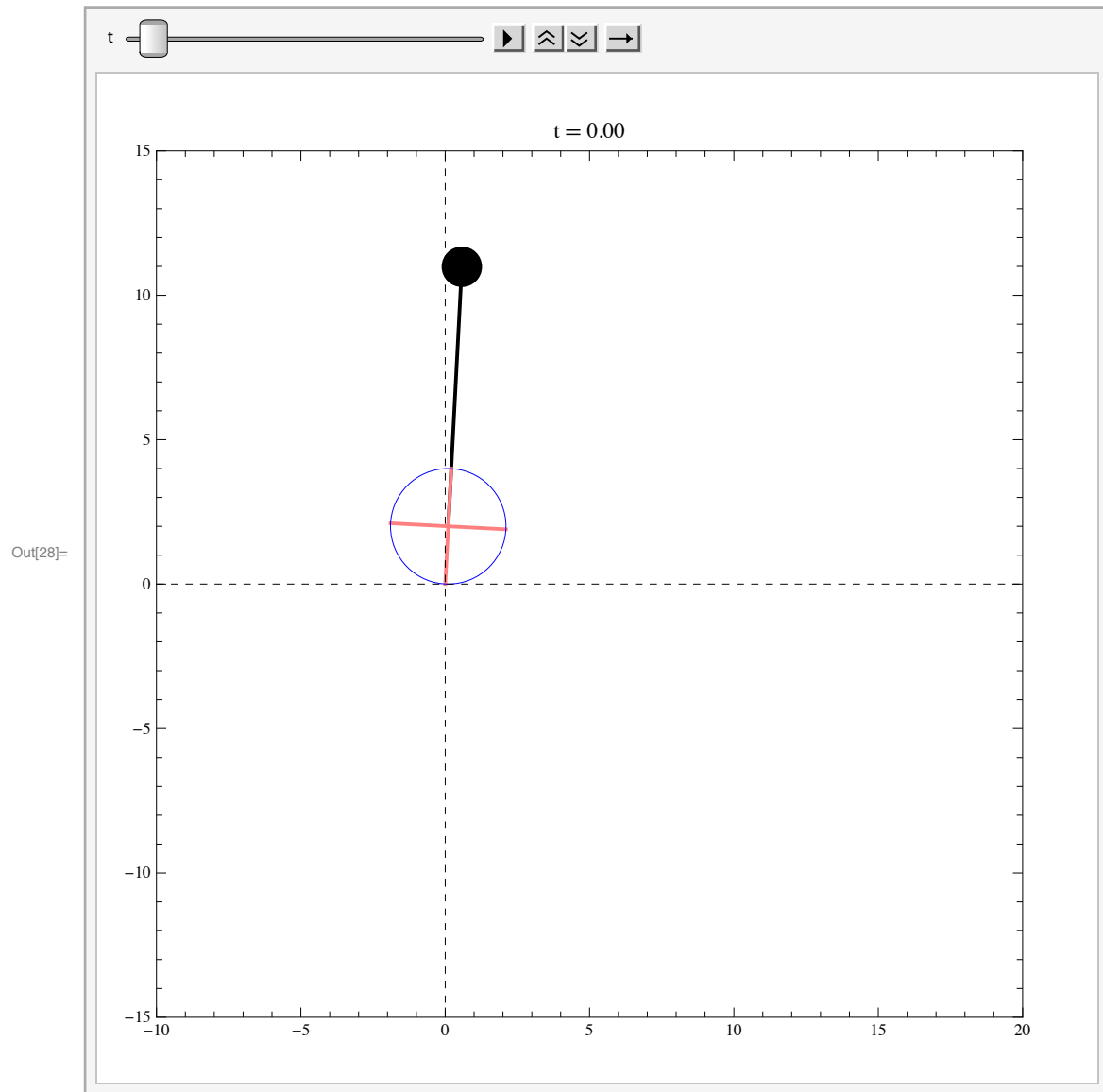
this equation describes the Segway body (relating $\theta(t)$ to $\beta(t)$)

$$l \sin(\theta(t)) (g + 2 r \dot{\theta}(t)^2) = \theta''(t) (l^2 + 3 l r \cos(\theta(t)) + 2 r^2) + r \beta''(t) (l \cos(\theta(t)) + 2 r)$$

■ Linearized Result

$$\frac{\Theta(s)}{B(s)} = \frac{-\frac{r}{l+r} s^2}{s^2 - \frac{g l}{(l+r)(l+2r)}}$$

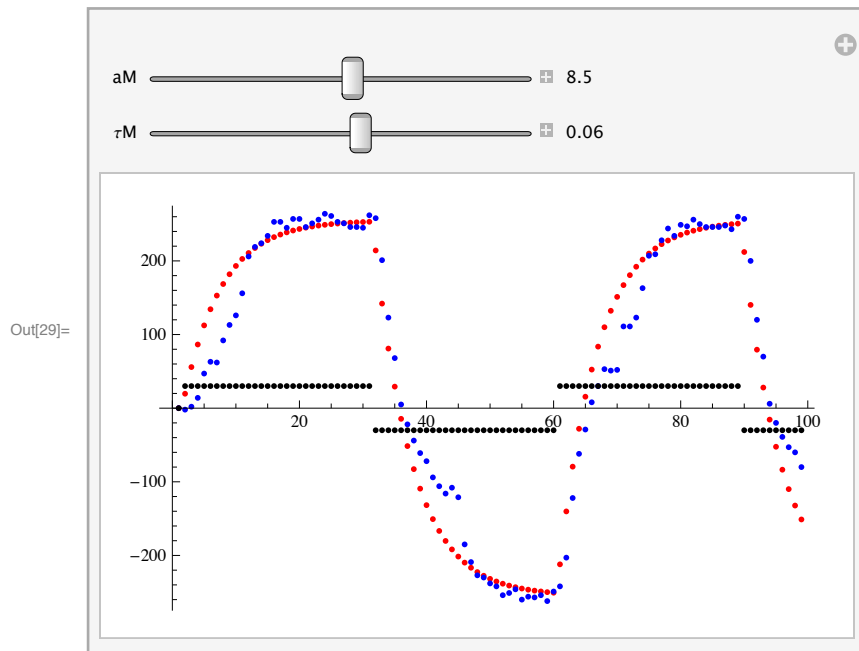
Simulation of Body Model (no feedback control)



Modeling of NXT Motor

■ Assume single pole response

- $\beta'(t) + \tau_M \beta''(t) = a_M \sigma(t)$ (this equation describes the motor)



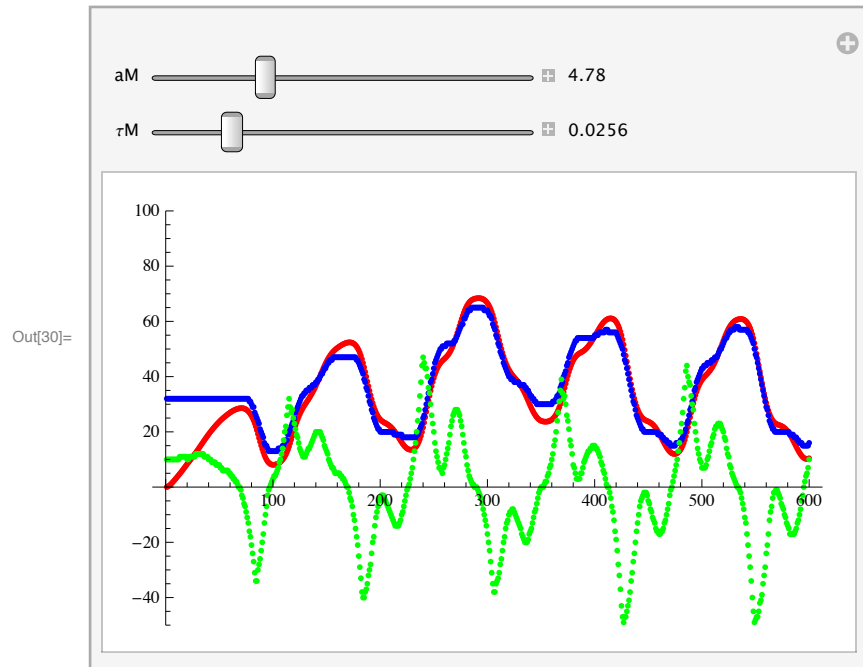
Unloaded motor output, $\dot{\beta}$ (deg/sec), 10 ms interval

Blue: Data

Red: Model

Black: Motor input, σ (30 unit step)

Modeling of NXT Motor (continued)



Motor output, β (deg), measurement under actual balancing conditions

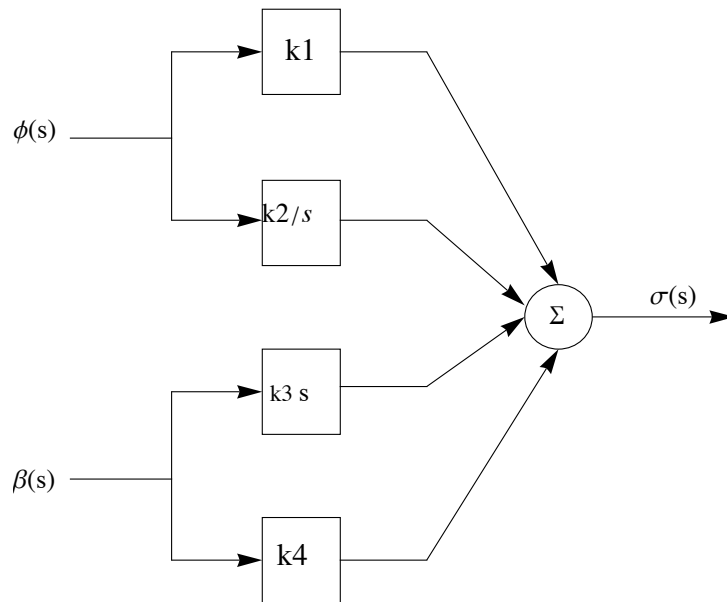
Blue: Data

Red: Model

Green: Input, σ

Controller Design

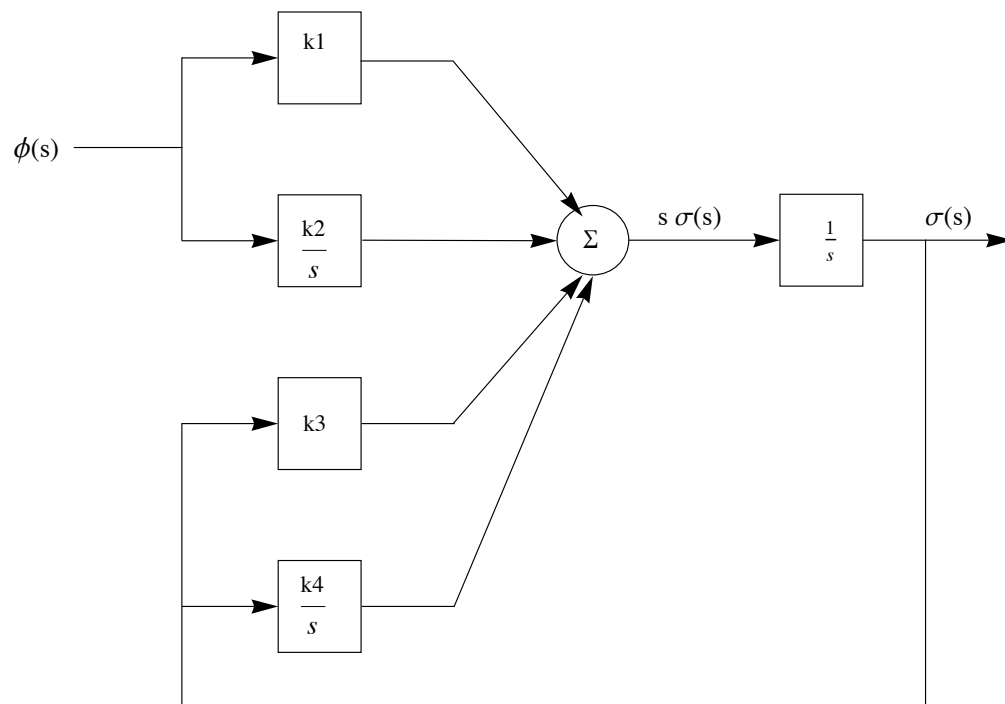
■ “NXTWay”



- $\sigma(s) = k_1 \phi(s) + \frac{k_2}{s} \phi(s) + k_3 s \beta(s) + k_4 \beta(s)$

Controller Design

■ “AnotherWay”

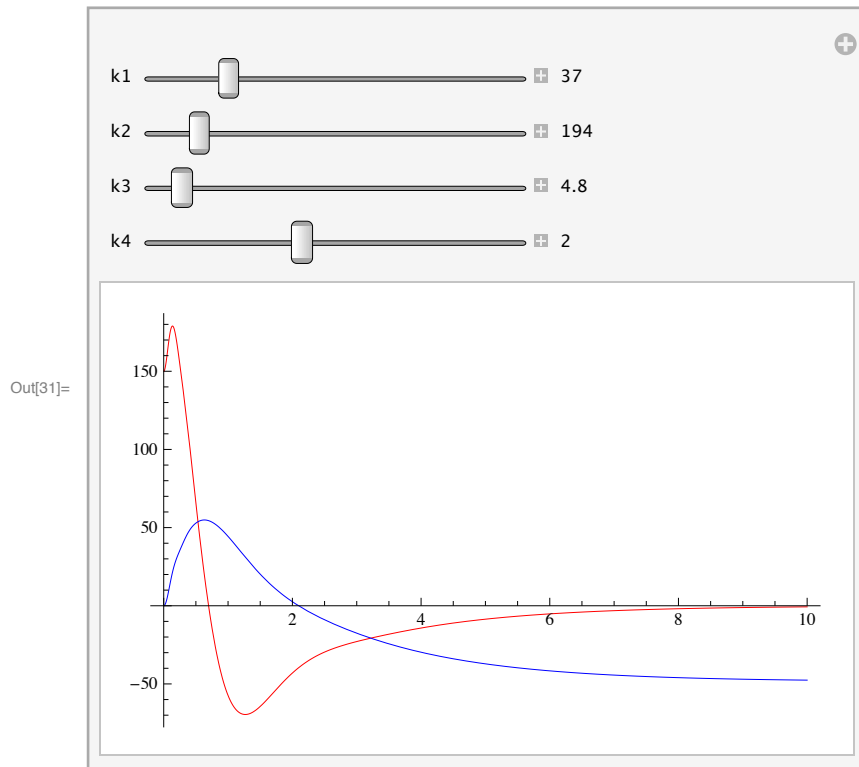


- $s \sigma(s) = k_1 \phi(s) + \frac{k_2}{s} \phi(s) + k_3 \sigma(s) + \frac{k_4}{s} \sigma(s)$

Controller Design (Continued)

- Simulation of differential equations describing the system...

$$\begin{aligned}
 & l \sin[\theta[t]] \left(g + 2 r \theta'[t]^2 \right) = \\
 & r \left(2 r + l \cos[\theta[t]] \right) \beta''[t] + \left(l^2 + 2 r^2 + 3 l r \cos[\theta[t]] \right) \theta''[t]; \\
 & \beta'(t) + \tau_M \beta''(t) = a_M \sigma(t); \\
 & \phi(t) = \theta'[t] + \text{offset}; \\
 & \sigma''[t] = k_1 \phi'[t] + k_2 \phi[t] + k_3 \sigma'[t] + k_4 \sigma[t];
 \end{aligned}$$

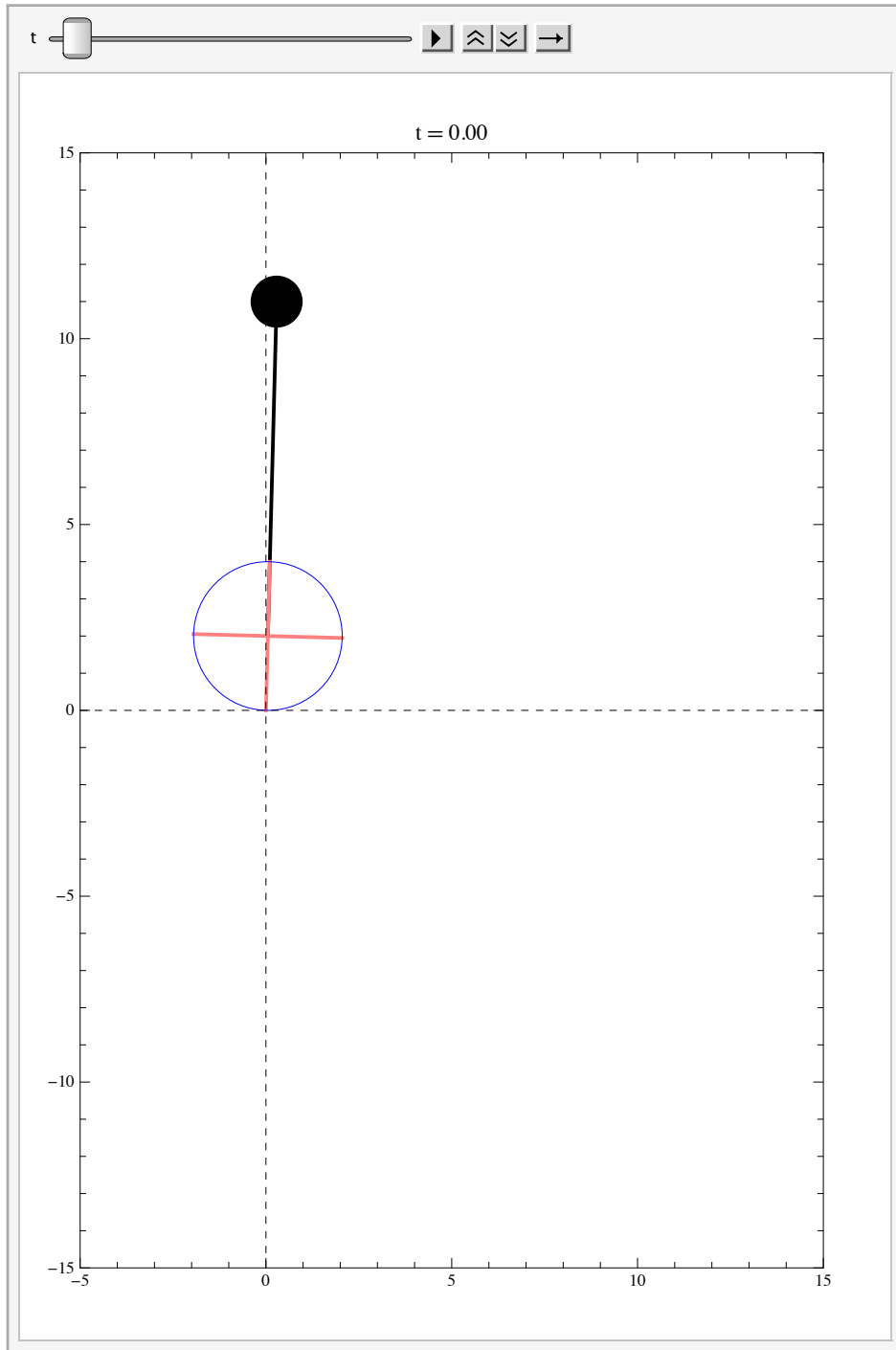


Continuous time simulation, .5deg/sec gyro offset, -1.5 deg/sec $\dot{\theta}$ offset

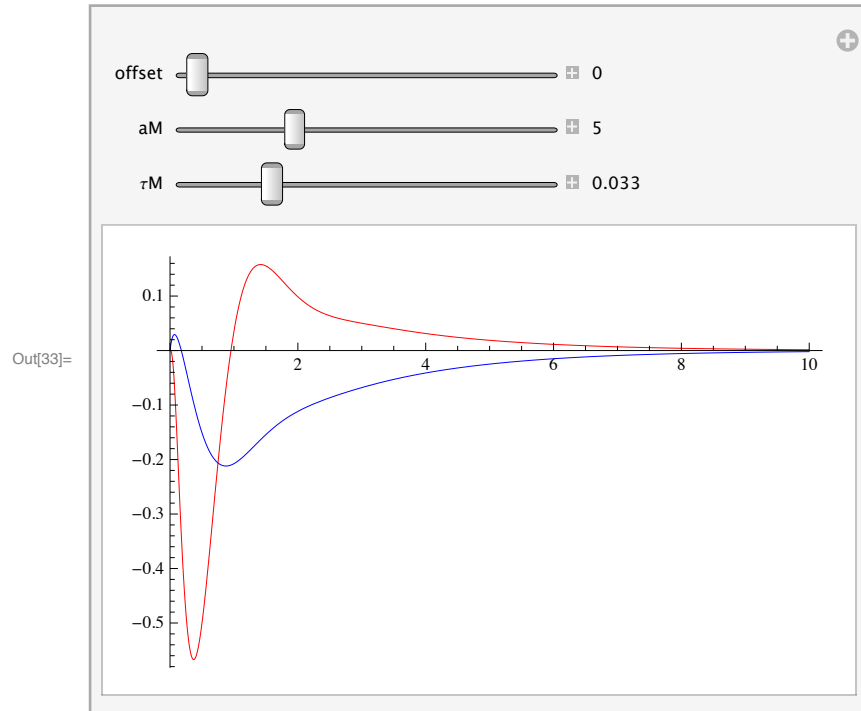
Red: Tilt angle, θ (x 100)

Blue: Controller output, σ

Continuous Time Simulation



Parameter Sensitivity Simulation



Red: Tilt angle, θ (x 100)

Blue: Controller output, σ

Digital Controller Implementation

- Differential Equation

$$\sigma''[t] = 194 \phi[t] + 37 \phi'[t] + 2 \sigma[t] + 4.8 \sigma'[t]$$

- Frequency Domain Gain

$$\frac{\sigma(s)}{\phi(s)} = \frac{194 + 37 s}{-2 - 4.8 s + s^2}$$

- Use software to convert to discrete time model

- Zero Order Hold method is good (also called step invariant transform)
- Mathematica function “ToDiscreteTimeModel”, Matlab function “c2d”

$$\frac{\sigma(z)}{\phi(z)} = \frac{-0.369389 + 0.389283 z}{1.05127 - 2.05148 z + z^2}$$

- Solve for latest output term

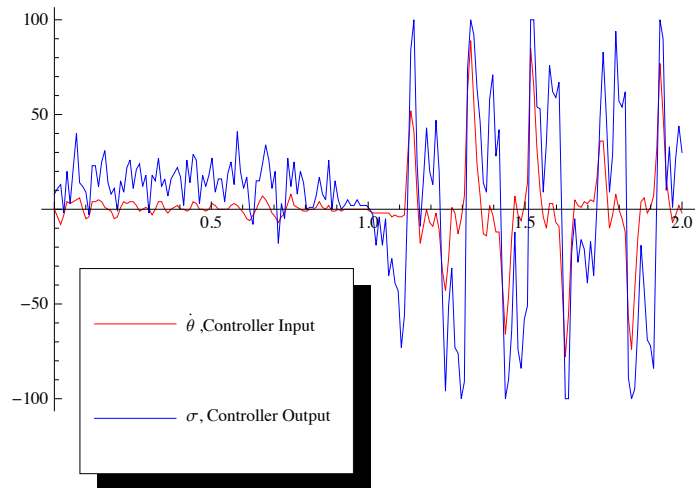
$$\sigma(z) z^2 = -0.369389 \phi(z) + 0.389283 z \phi(z) - 1.05127 \sigma(z) + 2.05148 z \sigma(z)$$

- Convert to time domain

This equation is implemented in the software

$$\sigma[n+2] = -0.369389 \phi[n] + 0.389283 \phi[n+1] - 1.05127 \sigma[n] + 2.05148 \sigma[n+1]$$

NXTWay Balancing



Conclusions

- A controller using positive feedback is a feasible solution for balancing a two wheeled robot.
- Changing motor direction causes significant nonlinearity and noise.
- Sampling time had to be increased from the design value to achieve balance due to probable inaccuracy from approximations.